

Optimization of Gasoline Engine Flywheel using Alternate Materials for Improving Performance Characteristics

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Abstract

Background/Objectives: The present investigation guides through the design, analysis and material selection process of Subaru EJ25D gasoline engine flywheel to reduce the weight of the flywheel without compromising its performance. **Methods/Statistical Analysis:** The flywheel was modeled using 3D modeling software Solid works. The validation of the strength of the flywheel was done by calculating and applying the forces on the flywheel using ANSYS software. The Finite Element Analysis (FEA) was conducted by considering the centrifugal forces acting on the flywheel. Structural analysis of the flywheel was done to validate the use of the alternate materials for constructing the flywheel. A comparative analysis of the stresses, total deformation and safety factor of the flywheel made of cast iron, titanium alloy, E-glass and aluminium alloy was reported. **Findings:** Based on the analysis, Aluminium alloy was found to be the most suitable material for constructing the flywheel. Specific tensile strength was considered for selection of the alternate materials of the flywheel. This paper also demonstrated the feasibility of using non-conventional materials like composite for manufacturing flywheels. **Applications/Improvements:** Flywheels which are a commonly used part in automobiles are subjected to multiple varying stress cycles which ultimately can lead to its failure.

Keywords: Alternate Materials, Engine, Flywheel, Gasoline, Performance

1. Introduction

The purpose of the flywheel is to smooth out power output due to the four stroke cycle of the IC engine. Flywheels are most commonly used in automobiles and are subjected to multiple varying stress cycles which ultimately can lead to its failure. They are also used in riveting machines to store energy from the motor and use it further during riveting operation. Flywheel is a mechanical device which is used as a storage device for rotational energy due to its significantly high moment of inertia¹. Flywheels are required where there is a fluctuation in input power and output load is constant or where there is a fluctuation in output load and the input power remains constant². Three factors account for the performance of the flywheel, namely

material strength, rotational speed and geometry. The kinetic energy level produced by the flywheel and its specific energy is determined by its material strength³. The major advantage is the ability of the flywheel to handle high power levels. In automobiles large peak power is necessary during acceleration and with the use of electrical breaks; a large amount of energy is generated for a short while during braking which results in lower fuel consumption.

They demonstrated that by using optimization techniques various parameters like material and cost can be optimized and by applying an approach for modification of various working parameters like efficiency, output and energy storing capacity, the results were compared to the existing flywheel of material Grey Cast Iron ASTM30⁷.

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Akshay et al. analyzed a flywheel with Finite Element Analysis. A structural analysis was performed and normal stress and radial deformation was calculated. Flywheel was discretized into 19735 elements and 36539 nodes. S-Glass Epoxy can be used in flywheels to store energy with less mass. It can also be used in high speed applications⁹. S.M. Arnold made a deformation analysis of composite flywheel disc systems. The two major aspects of failure analysis, that is static and cyclic limit (burst speeds) were addressed. In case of static limit loads, a lower bound for disks with constant thickness was presented for the case of pure rotation as in case of a free spinning disk.

2. Materials and Methods

The flywheel was modeled using Solidworks 2013. The structural analysis of the flywheel made of cast iron was carried out using ANSYS Workbench 15.0. The materials used for the analysis were Al 6063-T6, Titanium alloy and E-glass and the structural analysis was carried out. The results were compared to obtain the best material suitable for the flywheel. The methodology has been shown in Figure 1A.

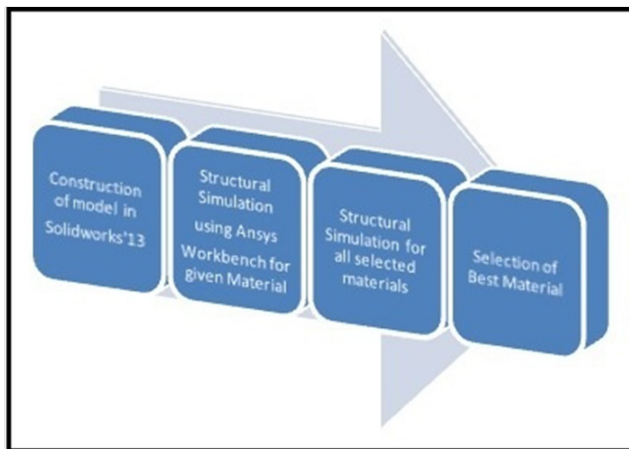


Figure 1. Methodology.

The selection of the materials was based on the ratio of tensile strength to the material density, also known as the specific tensile strength. The flywheel material with the highest specific tensile strength will yield the highest energy storage per unit mass¹¹. The materials were selected using the graph as shown in Figure 1.

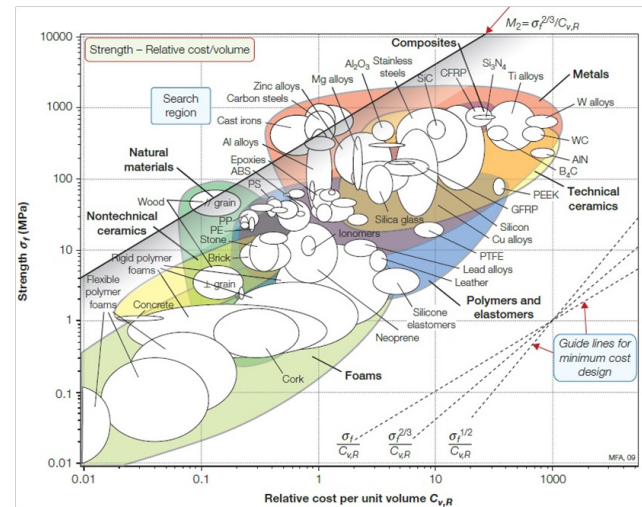


Figure 2. Graph of strength vs. density.

2.1 Specifications of Selected Engine

The flywheel of Subaru EJ25D gasoline engine from Forester 98 was selected for the analysis. The displacement, maximum power, maximum torque, number of cylinders, bore, stroke and compression ratio for the engine has been listed in Table 1.

Table 1. Engine specifications

Model	Forester 98
Displacement	2457 cc
Maximum power	115 hp @ 5600 rpm
Maximum torque	190 Nm @ 2800 rpm
Number of cylinders	4
Bore	99.5mm
Stroke	79mm
Compression ratio	9.7:1
Volume per cylinder	614.25cc

2.2 Modelling of the Flywheel

2D or 3D modeling can be carried out for constructing the flywheel. However, 3-D modeling was used to construct the flywheel as it gives more accurate results as compared to 2D modeling⁹. Figure 2 shows the dimensions of the Subaru flywheel. Figure 3 shows the exploded view of the 3D assembly with the flywheel, flexplate and spacer arranged from left to right.

2.3 Analysis of the Flywheel using FEA

ANSYS Workbench 15.0 was used for the analysis. The following assumptions were made for conducting the analysis:

An exploded view diagram of a planetary gear set. It consists of four main components: a central sun gear (small, silver), a planet carrier (large, silver, with multiple mounting holes), planet gears (small, silver, mounted on the planet carrier), and a ring gear (large, yellow, with internal teeth). The components are shown in their relative positions as they would be assembled.

Parameter/ Material	Cast iron	Al 6063 T6	Titanium alloy	E-Glass
Young's Modulus (GPa)	110	69	120	8.50
Ultimate tensile Strength (MPa)	448.08	240	1220	2050
Yield Strength (MPa)	241.2	215	1034.2	2050
Density (g/ cm^3)	7.3	2.7	4.8	2.6
Poisson's Ratio	0.26	0.33	0.33	0.23

3.1 Radial Deformation Analysis

The following trend was observed for the radial deformation:

$$\text{Al} < \text{Cast iron} < \text{Ti} < \text{E-glass}$$

The maximum deformation due to centrifugal force occurs in E-glass and the minimum in Al alloy, the reason for which can be contributed to their respective strength to the density ratio which was found to be the highest in case of Al and lowest for E-glass. The yield strength of Al is relatively less among the four, so the elastic modulus determines such behavior of aluminium alloys. The radial deformation is found to be maximum near the periphery and minimum near the center of the flywheel. This is because the centrifugal forces increase as we move towards the outer periphery of the flywheel from the center. Figure 4, Figure 5, Figure 6 and Figure 7 show the radial deformation for the different materials employed in structural simulation of the flywheel.

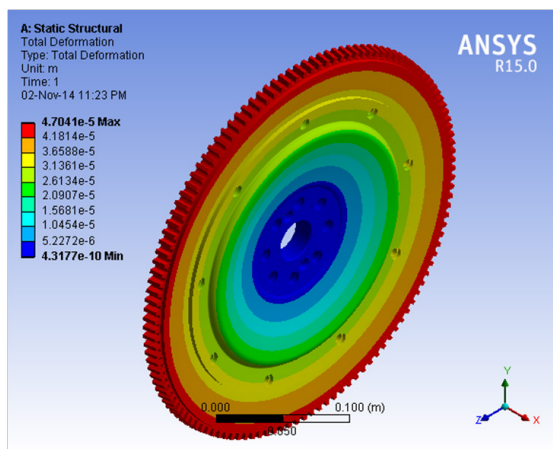


Figure 5. Radial deformation of cast alloy steel flywheel.

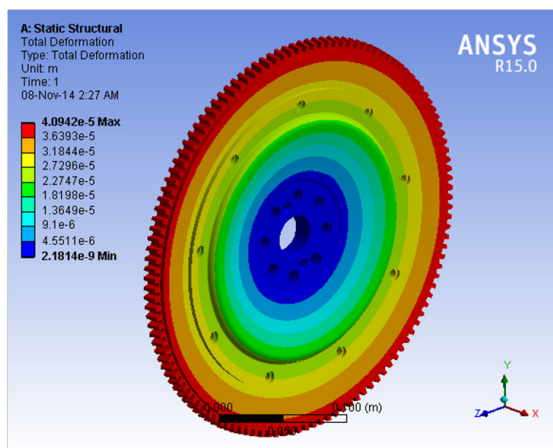


Figure 6. Radial deformation of Al 6063 T6 flywheel.

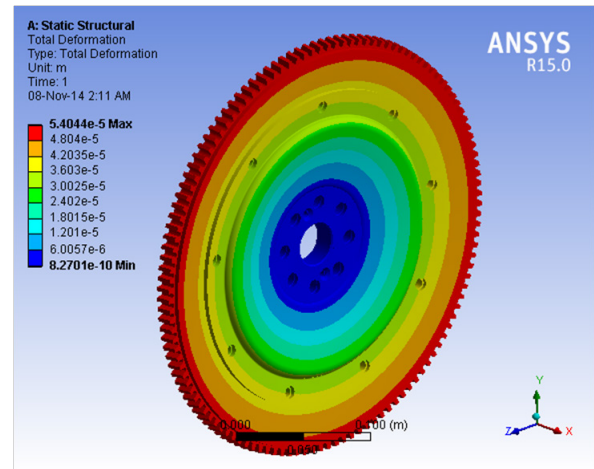


Figure 7. Radial deformation of Ti alloy flywheel.

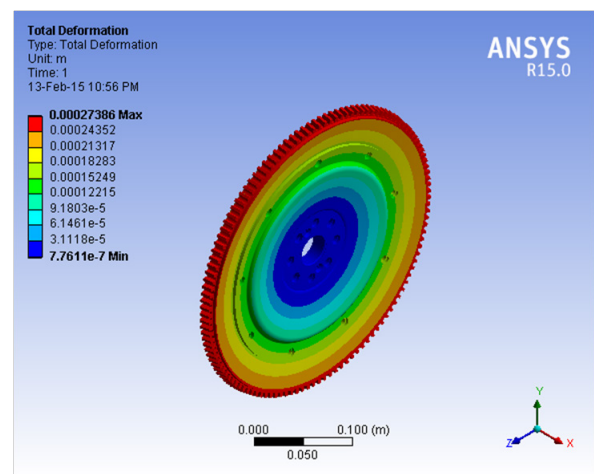


Figure 8. Radial deformation of E-glass flywheel.

3.2 Radial Stresses Analysis

The following trend was observed for radial stress analysis:

$$\text{Al} < \text{Ti} < \text{Cast iron} < \text{E-glass}$$

The maximum stress occurs in E-glass flywheel while the minimum occurs in Al alloy flywheel. One of the reason for this behavior can be attributed to the low strength to the density ratio for E-Glass and in the case of the aluminium, it can be proposed that as the strength to the density ratio is sufficiently high and the properties of aluminium alloy like light weight, high strength and high value of elastic modulus contribute towards minimum radial stress. In general, stiffer and lighter design can be achieved using aluminium alloys. Figure 8, Figure 9, Figure 10, and Figure 11 depict the radial stress for the different materials employed in structural simulation of the flywheel.

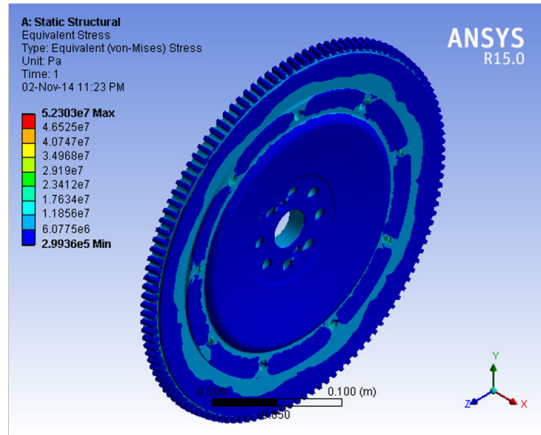


Figure 9. Radial stress developed in cast alloy flywheel.

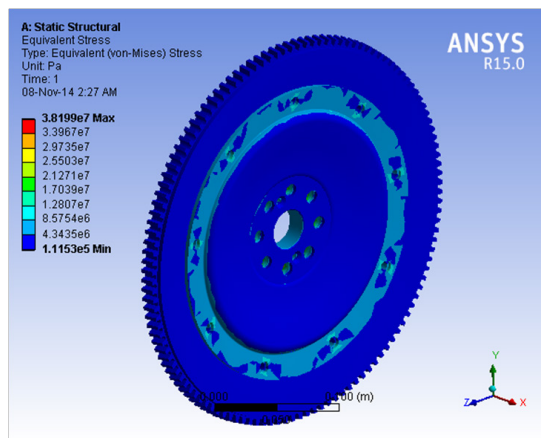


Figure 10. Radial stress developed in Al 6063 T6 flywheel.

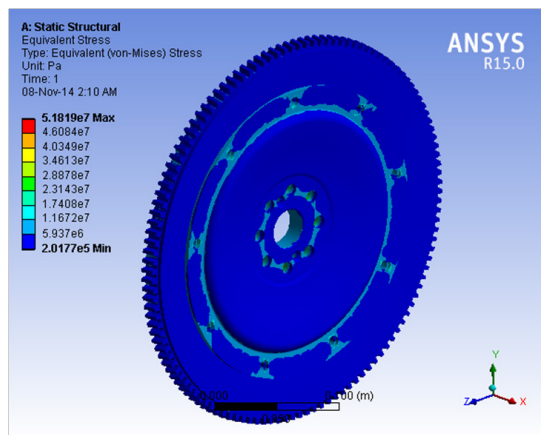


Figure 11. Radial stress developed in Ti alloy flywheel.

3.3 Factor of Safety Analysis

The factor of safety for the different materials of the flywheel are found to be within safe limits, which depicts

that the flywheel designs are feasible and their manufacturing can be carried out for real time simulation. Figure 12, Figure 13, Figure 14, and Figure 15 shows the factor of safety for the different flywheels.

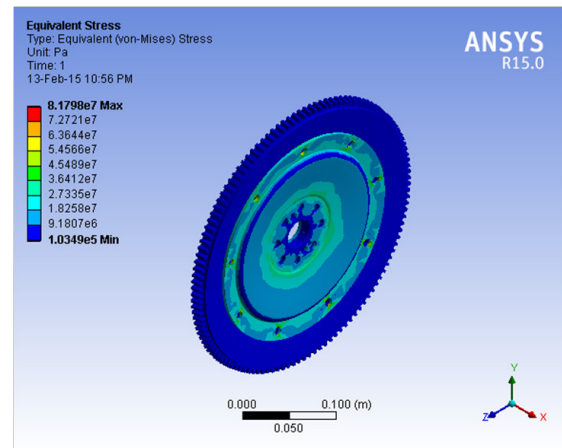


Figure 12. Radial stress developed in E-glass flywheel.

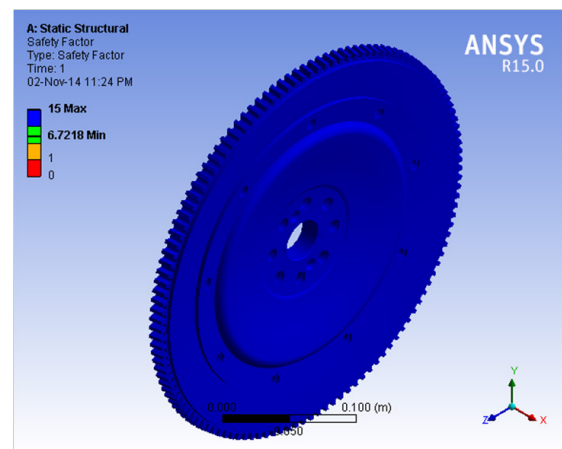


Figure 13. Factor of safety of cast alloy flywheel.

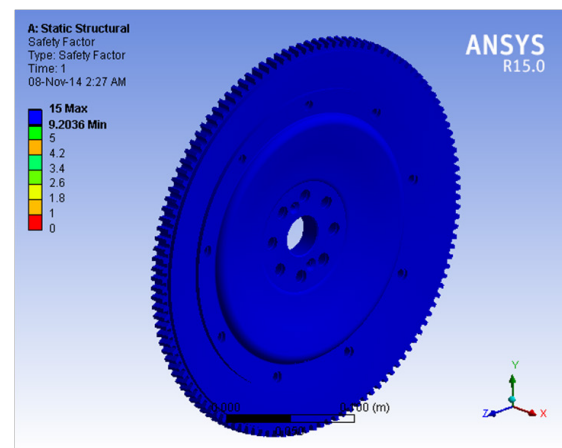


Figure 14. Factor of safety of Al 6063 T6 flywheel.

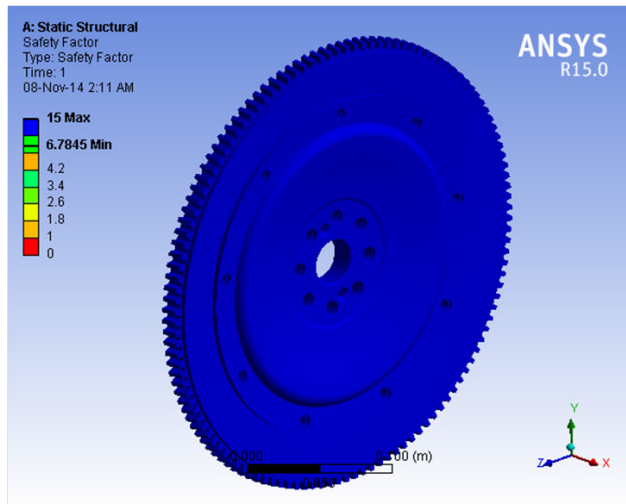


Figure 15. Factor of safety of Ti alloy flywheel.

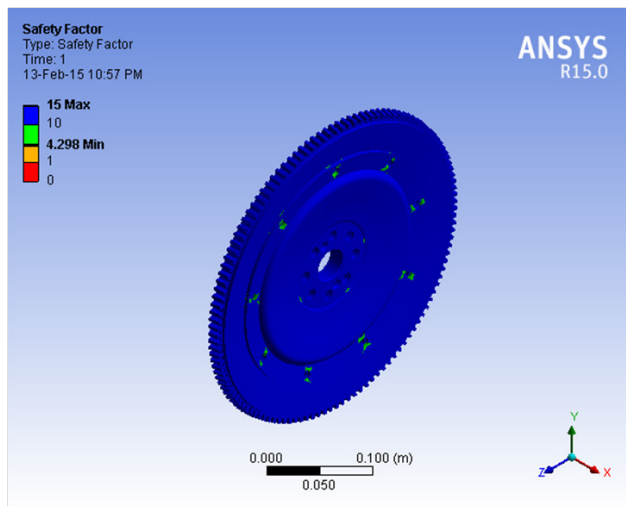


Figure 16. Factor of safety of E-glass flywheel.

The cast iron flywheel has the maximum weight while E-glass flywheel has the minimum weight. The results of the Finite Element Analysis (FEA) are listed in Table 3.

Table 3. Physical properties, radial stresses and deformation

Parameter/Material	Cast iron	Al 6063 T6	Titanium Alloy	E-Glass
Mass of Flywheel (Kg)	9.65	4.602	6.93	4.4924
Minimum radial Stress (MPa)	0.29	0.11	0.201	0.103
Maximum radial Stress (MPa)	52.3	38.1	51.81	81.97
Minimum Total Deformation (m)	4.3×10^{-10}	2.181×10^{-9}	8.27×10^{-10}	7.76×10^{-7}
Maximum Total Deformation (m)	4.7×10^{-5}	4.09×10^{-5}	5.4×10^{-5}	2.74×10^{-4}
Minimum Factor of Safety	6.7218	9.2036	6.7845	4.298

4. Manual Calculations

Formula Used

$$\sigma_{t1} = \frac{3}{4} * \rho * V^2$$

σ_{t1} = Tensile force due to centrifugal force acting on rim.

ρ = Density of material of rim

V = Linear velocity of rotation = $\frac{\pi * D * N}{60}$

Calculation of Linear Velocity (V)

Diameter of flywheel rim = $D = 0.314\text{m}$

Speed of flywheel = $N = 4000\text{rpm}$

$$V = \frac{\pi * D * N}{60} = \frac{\pi * 0.314 * 4000}{60} = 65.76\text{m/sec}$$

Calculation of tensile force (σ_{t1})

For Al-6063-T6

$$\sigma_{t1} = \frac{3}{4} * \rho * V^2$$

$$\sigma_{t1} = \frac{3}{4} * 2700 * 65.76^2$$

$$\sigma_{t1} = 87,56,864.64 \text{ Pa}$$

Similarly, the values for the cast iron, Ti alloy and E-glass are obtained and are shown in the table 4.

Table 4. Material Properties

Material	Al-6063C	Cast iron	Ti Alloy	E-Glass
Density (ρ), (kg/)	2700	8900	4820	2600
Tensile Strength ,(Pa)	87, 56, 864.64	288, 65, 220.4	156, 32, 625.02	84, 32, 535.32

5. Conclusion

The modeling of the flywheel was performed using Solid-works 2013. The Finite Element Analysis (FEA) was carried

out using ANSYS workbench 15.0 for the four materials: cast iron, Al 6063-T6, Titanium alloy and E-glass.

The structural analysis of the original flywheel made of cast iron was carried out and the deformation, radial stress and the Factor of Safety (FOS) was reported. The material of the flywheel was changed to Al 6063-T6, Titanium alloy and E-glass in succession, and the structural analysis on ANSYS workbench was reported for each of the materials. The results of the deformation and stress analysis were found to be within safe limits. The manual calculations are also found to agree with the results obtained from the analysis. However, the deformation and stress developed for aluminium 6063-T6 flywheel was found to be the least among the four materials under consideration. Al alloy also has an added advantage of higher strength-to-weight ratio compared to the other materials. Thus, Al was found to be the best suited material for constructing the flywheel. Thus by using Al alloy flywheel, 52.31% of the weight can be reduced as compared to cast iron flywheel which can help in reducing the weight of the vehicle and increase the fuel economy. However, the life of the flywheel will also get affected.

This paper also highlights the importance of composites as an alternative for constructing the flywheel due to its high specific tensile strength. This is one reason why E-glass is a material of interest. Using E-glass will result in a weight saving of 53.44% as compared to the cast iron flywheel. However, the radial stresses and deformation are considerably more than that of cast iron which makes it unsafe for use with the engine. Further study can be done to identify composites which can be feasible for manufacturing flywheels with the help of structural analysis.

6. References

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